AMENDMENTS TO THE SPECIFICATION WITH MARKINGS TO SHOW CHANGES MADE

Amend the following paragraph(s):

[0032] -- FIG. 4 shows a half-section of still another embodiment of a bearing arrangement according to the present invention. [[.]]Parts that differ from the parts shown in FIG. 1 will be identified by corresponding reference numerals followed by a "b". In this embodiment, the bearing arrangement has a resistance member 10b of two part construction, with one part formed by the axial projection 11b which, in analogy to the embodiment of FIG. 3, is press-fitted in the bore 22 of the base 14 of the trunnion 1 and supported upon the abutment area 29 of the gradation 21. The other part of the resistance member 10b is formed by a ring element 23 which is placed against the inside of the wall 5 in a centered manner to provide an axial stop for the rolling elements 7. The lubricant reservoir is provided here by the ring space 15 in the trunnion 1 and an intermediate space 24 which is defined between the bottom wall 9 of the cup 2 and the end surface 13 of the trunnion 1. The intermediate space 24 has hereby a width which is characterized by the gap s. Thus, an axial force acting upon the trunnion 1 in the direction of arrow A is first absorbed by the projection 11b and transmitted via the gradation 21 into the bottom wall 9 of the cup 2. A further increase in the axial force results in an elastic deformation of the projection 11b until the end surface 13 of the trunnion 1 is forced against the bottom wall of the cup 2.--

[0033] -- FIG. 5 shows a half-section of still another embodiment of a bearing arrangement according to the present invention. [[.]]Parts that differ from the parts shown in FIG. 1 will be identified by corresponding reference numerals followed by a "c". In this embodiment, the trunnion 1 has a cylindrical end surface 13 so that the lubricant reservoir is formed entirely by the recess 17. The bearing arrangement has a resistance member 10c which has a central axial projection [[11e]] 11c which is of shorter axial length and supports the end of the

trunnion 1 in the area of the end surface 19. Thus, the size of the recess 17 is directly impacted by the axial length of the projection 11c.--.

[0034] -- FIG. 6 shows a half-section of still another embodiment of a bearing arrangement according to the present invention. [[.]]Parts that differ from the parts shown in FIG. 1 will be identified by corresponding reference numerals followed by a "d". In this embodiment, the outer cup 2 has a bottom wall 9d of stepped configuration to define a step 25 of smaller inner diameter and a step 26 of greater inner diameter. The bearing arrangement includes a resistance member 10d which is centered against the step 25 and has a central axial projection 11d of a length l which is greater than a height h of the stepped configuration. This assures a guiding of the trunnion 1 on the end surface 19 of the projection 11d, and that the end surface 13 of the trunnion 1 is supported on the step 26 of the bottom wall 9 only when the axial force exceeds a certain level and the projection 11d has undergone an elastic deformation. The lubricant reservoir is again formed solely by the recess 17 which is demarcated in radial direction by the outer surface area of the projection 11d and the bottom wall 9 in the area of the step 25.--.

[0035] -- FIG. 7 shows a half-section of still another embodiment of a bearing arrangement according to the present invention. [[.]]Parts that differ from the parts shown in FIG. 1 will be identified by corresponding reference numerals followed by an "e". In this embodiment, the bearing arrangement has a resistance member 10e which is characterized by the absence of a centered axial projection but includes only a ring element to form the collar 16 centered at the wall 5 of the cup 2 and abutting against the rolling elements 7. Extending radially inwardly from the collar 16 are the stop members 18 in analogy to the embodiment of FIG. 1 so as to provide the support of the trunnion 1 when the axial force on the trunnion 1 is at a certain level. The cup 2 is formed with the gradation 21 for support of a central flange 27 extending axially from the end surface 13. The lubricant reservoir is

formed here by the ring-shaped space 17 demarcated by the collar 16, flange 27 and the end surface 13 of the trunnion 1 and having a volume directly dependent on the axial length of the flange 27.--.

[0036] -- FIG. 8 shows a half-section of still another embodiment of a bearing arrangement according to the present invention. [[.]]Parts that differ from the parts shown in FIG. 1 will be identified by corresponding reference numerals followed by an "f". In this embodiment, the bearing arrangement has a resistance member 10f having a central axial projection 11f of short axial extension for support of the flange 27 of the trunnion 1. The lubricant reservoir is formed here by the substantially circular ring-shaped space 17 of a volume determined by the axial lengths of the projection 11f and the flange 27. In addition to the stop members 18, the resistance member 11f includes further stop members 28 of different axial length than the stop members 18 so that different gap sizes s_1 and s_2 are defined between the end surface 13 of the trunnion 1 and the resistance member 11f. This construction effects a multi-indexed support of the trunnion 1 upon the resistance member 10f and the bottom wall 9 of the cup in dependence on the applied axial force.--.